Creo Simulate 4.0
Structural and Thermal Analysis

Learning Guide
1st Edition
AS-CRS4-STA1-SG   //   RS-CRS4-STA1-SG

ASCENT - Center for Technical Knowledge®
Creo Simulate 4.0
Structural and Thermal Analysis
1st Edition

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Preface

This learning guide covers the fundamentals of Creo Simulate 4.0: Structural and Thermal Analysis. It provides you with the knowledge to effectively use Creo Simulate for finite element analysis, thereby reducing design time. Many concepts apply to both Structure and Thermal analysis; a portion of this guide is specifically dedicated to Thermal analysis. This is an extensive hands-on learning guide, in which you have the opportunity to apply your knowledge through real-world scenarios and examples.

Topics Covered

• FEA Fundamentals: P-elements and analysis convergence methods
• Basic Modeling and Analysis
• Types of Loads and Constraints
• Idealizations: Shells and Beams
• Sensitivity and Optimization Studies
• Assembly Interfaces and Contact Analysis
• Thermal Analysis
• Modal Analysis
• Welds, Springs, and Masses
• Fasteners and Rigid Links
• Buckling Analysis

Note on Software Setup

This learning guide assumes a standard installation of the software using the default preferences during installation. Lectures and practices use the standard software templates and default options for the Content Libraries.

This guide was developed against the M020 builds of Creo Parametric 4.0 and Creo Simulate 4.0.
In this Guide

The following images highlight some of the features that can be found in this Learning Guide.

Practice Files

The Practice Files page tells you how to download and install the practice files that are provided with this learning guide.

Chapters

Each chapter begins with a brief introduction and a list of the chapter’s Learning Objectives.

Learning Objectives for the chapter

Getting Started

In this chapter you learn how to start the AutoCAD® software, become familiar with the basic layout of the AutoCAD screen, how to access commands, use your pointing device, and understand the AutoCAD Command window. You also learn how to open an existing drawing, view a drawing by zooming and panning, and save your work in the AutoCAD software.

Learning Objectives for this Chapter

- Launch the AutoCAD software and complete a basic setup of a drawing environment.
- Identify the basic screen and functions of the AutoCAD drawing window, the Ribbon, Drawing Window, and Application bar.
- Locate commands and launch them using the Ribbon, shortcut menus, Application Menu, command line, and keyboard.
- Locate points in the AutoCAD Command window.
- Open and close existing drawings and manage drawing locations.
- Move around in a drawing using the mouse, the Zoom and Pan commands, and the Navigator bar.
- Save drawings in various formats and set automatic save options using the Save options.
Side notes
Side notes are hints or additional information for the current topic.

Practice Objectives

Practice 1c

Estimated time for completion: under 5 minutes.

Saving a Drawing File

Previous Objectives
- Close and save a drawing
- Modify the AutoSave settings

In this practice, you will open a drawing, save it, and modify the AutoSave settings, as shown in Figure 1-31.

Figure 1-31

1. Open Building Valley.dwg from your class folder.
2. In the QMenu, click File | Save. In the Command Line, QSAVE displays indicating that the AutoCAD software has performed a quick save.
3. In the Application Menu, click QSAVE to open the Options dialog box.
4. In the QuickSave tab, change the time for Automatic save to 15 minutes.

Instructional Content

Each chapter is split into a series of sections of instructional content on specific topics. These lectures include the descriptions, step-by-step procedures, figures, hints, and information you need to achieve the chapter's Learning Objectives.

Practices

Practices enable you to use the software to perform a hands-on review of a topic.

Some practices require you to use prepared practice files, which can be downloaded from the link found on the Practice Files page.
Chapter 1

Introduction to Creo Simulate

Creo Simulate is a powerful software tool that enables you to simulate structural and thermal behavior of your design to understand and improve the design’s performance.

Learning Objectives in this Chapter

• Understand the concept of FEA.
• Understand the concepts of H-refinement and P-refinement.
• Understand the advantages of P-elements.
• Understand solution convergence methods in Creo Simulate.
• Understand the analysis abilities of Creo Simulate.
• Understand the steps involved in a Creo Simulate analysis.
• Understand the recommendations for CAD model preparation.
• Understand the Creo Simulate modes of operation.
1.1 Finite Element Analysis (FEA)

Finite Element Analysis is a numerical mathematical method based on the following process:

- Discretize (i.e., divide) the model into smaller and more simplified volumes (tetrahedra, bricks, wedges, etc.) called finite elements. The collection of finite elements approximates the shape of the model, and is called finite element mesh, or just mesh. An example of a meshed model is shown in Figure 1–1.

- Approximate the variation of the principal quantity of interest (such as displacement, stress, etc.), within each finite element with polynomials. These polynomials are typically called local approximation functions or shape functions.

- Connect the finite elements across the inter-element boundaries, thus effectively sewing elemental polynomials together. The sewn local polynomials now approximate a variation of the quantity of interest over the entire model, and therefore comprise the global approximation function in the form of a piece-wise polynomial.

Creo Simulate contains the AutoGEM tool, which automatically meshes a model.

In 2D models, finite elements are triangles or quadrilaterals. In 3D models, finite elements are 4-node tetra, 6 node wedge or 8-node bricks.
• Solve the governing equations and boundary conditions for the global approximation function, and find the best fitting solution. In structural mechanics, the principle of minimum total potential energy is typically used to find the best fitting solution, which results in solving a large number (sometimes hundreds of thousands), of simultaneous linear equations.

• Present the results for this approximate solution.

Therefore, the key FEA concept is the use of piece-wise polynomials to approximate the sought field quantity in the model, which effectively replaces a continuum problem with an infinite number of degrees of freedom (DOF) by a discrete problem with a finite number of DOF (i.e., finite elements and discretization).

For example, consider how the FEA method works when applied to calculate deflections in a simple beam as shown in Figure 1–2. The beam is clamped at the left end, has a couple of supports in the middle, and is loaded by a couple of transversal forces and a moment. The bottom graph shown in Figure 1–2 represents the unknown true deflection of the beam, which you are trying to determine using the FEA method.

![Figure 1–2](image-url)
The first step in the process (shown in the example in Figure 1–3) is to mesh the beam by breaking it into a collection of shorter pieces (i.e., finite elements) connected at their ends (i.e., the nodes).

Next, the deflection $Y$ within each finite element is approximated by a polynomial. In this example, you use linear polynomial $Y = a_0 + a_1X$, which means that deflection within each element is approximated by essentially a straight line.

Next, the local linear polynomials are sewn together at the nodes, creating a global approximation function in the form of a piece-wise linear polynomial, which is a polyline.

Finally, the global approximation function is best-fit to satisfy both the bending differential equations and beam boundary conditions (loads and constraints). The resulting function (the dashed line shown in Figure 1–3) now represents the FEA solution for the true deflection (the solid line shown in Figure 1–3) in the beam.

It is important to note that your FEA result contains a certain amount of error, which is the deviation between the true deflection (the solid line shown in Figure 1–3) and the FEA solution (the dashed line shown in Figure 1–3), and which is called a discretization error.

Any FEA solution is just an approximation, which means it always contains a discretization error. Therefore, in the FEA process, it is critical to know how to estimate, how to control, and how to reduce this unavoidable approximation error to acceptable levels.

Note that sewing local polynomials at the nodes ensures continuity of the global approximation function, and therefore of the FEA solution for the deflection over the entire beam.
1.2 FEA Solution Refinement

The process of bringing the FEA approximation error to acceptable levels is typically called *solution refinement*. There are two alternative ways in which an FEA solution can be refined.

The first option involves making the finite elements in the mesh progressively smaller while maintaining the order of polynomials within each element.

For example, consider the beam shown in Figure 1–3. If you make the finite elements smaller without changing anything else, the approximation error becomes smaller as well, as shown in Figure 1–4.

![Figure 1–4](image)

This approach is called *h-refinement* because the letter *h* in FEA literature typically refers to the size of the finite elements in the mesh. It is also worth noting that h-refinement requires re-meshing the model every time you need a more accurate solution.

The h-refinement approach is used by most FEA software systems that are commercially available today. However, this is not the only available option.

An alternative strategy involves increasing the order of polynomials within the finite elements, without changing the elements' sizes.
Again, consider the example of the beam shown in Figure 1–3. If you use second-degree polynomials $Y = a_0 + a_1X + a_2X^2$ to approximate the deflection within each element, this results in a more accurate solution, without needing to make the finite elements smaller, as shown in Figure 1–5.

![Figure 1–5](image)

This approach is called \textit{p-refinement}, because the letter \textit{p} in FEA literature typically refers to the order of polynomials, and the process of progressively increasing the polynomial order is called \textit{polynomial escalation}.

The \textit{p-refinement} mathematical apparatus has been historically developed much later than the \textit{h-refinement}. Today, the \textit{p-refinement} approach is only used by a few commercial FEA software systems, one of which is Creo Simulate.
1.3 P-Elements

Creo Simulate exclusively uses p-elements and p-refinement technology to ensure the accuracy of the solution. The maximum polynomial order in Creo Simulate can be as high as 9. (The maximum polynomial order in the h-version of FEA is typically 2.)

The advantages of p-technology over h-technology are as follows:

• Solution accuracy can be improved without having to re-mesh the model.

• P-elements use hierarchical polynomials, which permits the use of different polynomial orders in different areas of the model for better efficiency.

• The rate of convergence to the true solution is greater than that of the h-technology.

• High stress gradients, such as in stress concentrators, are simulated extremely well.

• The restrictions on the shape of elements (aspect ratio, skewness, etc.) are less stringent. Therefore a p-mesh always contains fewer elements that an h-mesh.

• P-elements have curvilinear boundaries and tend to approximate CAD geometry very well.

*The rate of convergence refers to how quickly the refinement process converges to the true solution.*
1.4 Convergence Methods

Convergence in FEA (also called adaptivity) is a process of automatic solution refinement to achieve the required accuracy. In other words, the FEA software automatically adapts the solution parameters to better fit the true solution.

One of the key advantages of p-refinement over h-refinement is that p-elements permit an adaptive solution improvement without re-meshing the model. Instead, the maximum orders of polynomials used to approximate the solution are increased as required. The solution process can then be repeated on the same mesh, with the new increased polynomial orders. Such an adaptive step (called pass in Creo Simulate) can be repeated until the required accuracy is achieved.

In p-elements, the polynomial orders (called P-levels in Creo Simulate) can be assigned independently to each edge, face, or solid in the mesh. Using the convergence algorithm, Creo Simulate can pick P-levels independently for each mesh edge in the model, the goal being to select just the correct P-levels to achieve the required solution accuracy at minimum computational expense.

Multi-Pass Adaptivity (MPA) is the most commonly used convergence method in Creo Simulate. To identify the edges that warrant a P-level increase, the MPA algorithm compares displacements and element strain energies on the current solution pass with the corresponding values on the previous pass. Where the difference is larger than the user-specified percentage (i.e., convergence percentage), the P-level is increased and otherwise left unchanged. This process is repeated until the user-specified convergence percentage for the solution is met. The convergence criteria might include percentages on the default local and global quantities, such as displacement, strain energy, and RMS stress, but could also involve user-defined solution parameters.
The MPA convergence graphs can be visualized once the analysis has finished, as shown in Figure 1–6.

Since convergence percentage(s) are selected by the user, the MPA algorithm provides the user with maximum control, and is best used if the accuracy of the solution is critical.

The second convergence algorithm in Creo Simulate, called Single-Pass Adaptivity (SPA), uses a different theoretical foundation to reach an accurate solution.

The SPA algorithm is based on the fact that, although displacements in an FEA solution are continuous between elements, stresses are not, and the magnitude of stress jump at the discontinuity is a good indicator of the solution accuracy (i.e., the greater the stress jump, the less accurate the solution).

In the SPA algorithm, Creo Simulate first calculates the solution for P-level 3, assigned uniformly to all edges, and average stress discontinuities around each element (i.e. element error indicators) are computed. The P-levels of edges belonging to elements with large stress jumps are increased. Edges of elements with larger errors receive a higher P-level increase than edges of elements with lower errors. The solution is then repeated, and the result obtained at this point is taken as the final answer. The element error indicators are recomputed to indicate the overall stress accuracy.

Since only two convergence passes are performed in SPA, the computation time is typically much shorter than in MPA.
No convergence graphs are available in the SPA algorithm. Instead, the RMS stress error estimate is printed out to the Creo Simulate report file, as shown in Figure 1–7.

<table>
<thead>
<tr>
<th>Load Set</th>
<th>Stress Error</th>
<th>% of Max Prin Str</th>
</tr>
</thead>
<tbody>
<tr>
<td>LoadSet1</td>
<td>1.26e+01</td>
<td>10.3% of 1.22e+02</td>
</tr>
</tbody>
</table>

**Figure 1–7**

The SPA algorithm has been optimized by PTC with the goal of obtaining as good or better result as using the MPA convergence with the default (10%) convergence percentage.

The SPA convergence method provides limited control over the accuracy of the solution. Therefore, it should be reserved for quick design-analysis iterations when solution accuracy is not critical.

The third convergence method in Creo Simulate is called **Quick Check**, and does not perform a convergence process. The model is only run once, with all of the P-levels fixed at 3. The results of a Quick Check should not be trusted. The intention of a Quick Check analysis is to quickly run the model through the solver to detect any potential modeling errors (such as in constraints), before committing to a more lengthy analysis run (such as when using MPA).
1.5 Types of Analysis in Creo Simulate

Creo Simulate analysis capabilities straddle two physics domains:

- **Structural**: Determines deformations, stresses, and strains in solid bodies caused by external forces, moments, and other types of loading.
- **Thermal**: Determines temperatures and heat fluxes in solid bodies due to heat sources and/or sinks.

The Structural part of Creo Simulate can perform the following types of analysis:

- Static (including nonlinear material models, large displacements, and contact)
  - Pre-stress Static
  - Buckling
  - Fatigue
- Modal (Natural Vibrations)
  - Pre-stress Modal
- Dynamic Time Response
- Dynamic Frequency Response
- Dynamic Random Response
- Dynamic Shock Response

The two options for the Thermal analysis are as follows:

- Steady State Thermal analysis
- Transient Thermal analysis

The models in Creo Simulate can be analyzed in 3D formulations (purely solids, or combinations of solids, shells, and beams) or 2D formulations (plane stress, plane strain, or axisymmetric).
1.6 FEA Process

A typical FEA analysis process in Creo Simulate consists of three principle steps, as shown in Figure 1–8:

- **Pre-processing**: All input data for the analysis is prepared, such as material properties, loads, and constraints.

- **Solution**: The convergence type and criteria are specified and the analysis computation is performed.

- **Post-processing**: The analysis results are reviewed and verified. A report is prepared.

Figure 1–8

The CAD model simplification step is optional. It might not be required, depending on the complexity of the model.
1.7 CAD Model Preparation

A CAD model is developed to provide detailed information for manufacturing. All of the required information related to fillets, rounds, holes, and threads must be included. Processing steps and surface finishes are indicated and dimensions are fully specified.

An FEA model is developed to determine model behavior under a specific set of loading and boundary conditions. To analyze a model effectively, an FEA model is often different from a model developed for manufacturing. The symmetry of a model can often be used. Minor features, such as rounds, fillets, chamfers, and holes, can often be ignored unless they have a large effect on the result. Therefore, the general recommendation is to use the simplest model possible that is going to yield reliable results at the lowest computational time and cost.

In the example shown in Figure 1–9, the area of interest is the stress in the weld between two pipes due to high pressure. The FEA model within the component is shown on the right. In this case, the symmetry of the component (1/2 of the component) is used for the FEA model. The minor rounds, fillets, chamfers, and holes are ignored. The CAD model prepared for FEA would be different if the area of interest was the stress at the intersection of lips and pipes.

Figure 1–9

Lips

Area of interest (blend)
1.8 Creo Simulate Modes of Operation

Creo Simulate can operate in the following modes:

- **Integrated**: The simulation is fully integrated within the Creo Parametric design process (i.e., the models can be analyzed without ever leaving the Creo Parametric user interface). This is the most common way to use Creo Simulate.

- **Standalone**: Enables the loading of CAD models directly into Creo Simulate, without first loading Creo Parametric. This is useful if models originating from different software than Creo Parametric need to be analyzed.

- **FEM**: Provides pre- and post-processing capabilities only. The model has to be exported to a neutral file format and then solved by a 3rd party FEA solver (ANSYS, NASTRAN, etc.). H-elements meshing is used.

- **Simulate Lite**: Limited model size of up to 200 surfaces and also has a limited user interface. Does not require a Simulate license.

This learning guide focuses on the Integrated mode, which provides the most streamlined approach to part or assembly simulation and optimization within the Creo environment.